# **Dynamic Loading**

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## **Static vs. Dynamic Loading**

• Static Loading: external loads are gradually increased to a definite value and subsequently are held in constant.



• **Dynamic Loading:** the magnitude and/or direction of external loads vary as a function of time. This type of loads generate significant acceleration within structural members that cannot be neglected.









#### **Structural Members under Constant Acceleration**

- Given: *P*, *A* and *a*.
- Find: the stress developed in the steel rope.
- Solution by the **method of statnamic**:
- 1. Inertia force:  $F = \frac{P}{a}$ 2. Axial force in rope:  $N_d = P + \frac{P}{g}a = P\left(1 + \frac{a}{g}\right)$ 3. Normal stress developed in rope:  $\sigma_d = \frac{N_d}{A} = \frac{P}{A}(1 + \frac{a}{q}) = \sigma_{st}\left(1 + \frac{a}{q}\right)$ 4. Dynamic load factor:  $k_d = \frac{N_d}{N_{st}} = \frac{\sigma_d}{\sigma_{st}} = \frac{\Delta_d}{\Delta_{st}} = \left(1 + \frac{a}{g}\right)$



#### **Structural Members under Constant Rotation**

- Given:  $\omega$ , A,  $\rho$  and D.
- Find: the stress developed in the flange of the flying wheel.
- Solution by the **method of statnamic**:
- 1. Inertia force acting on unit length:

$$\boldsymbol{q}_{d} = \boldsymbol{\rho} \cdot 1 \cdot \boldsymbol{A} \cdot \boldsymbol{\omega}^{2} \frac{\boldsymbol{D}}{2}$$

2. Circumferential force:

$$F_{\rm N} = \frac{1}{2} \int_0^{\pi} q_d \cdot \frac{D}{2} d\varphi \sin \varphi = \frac{\rho A D^2 \omega^2}{4}$$

3. Circumferential stress:

$$\sigma_d = \frac{F_{Nd}}{A} = \frac{\rho \omega^2 D^2}{4}$$



### **Horizontal Impact on Axial Members**



- Horizontal Impact: Consider a rod which is hit at its end with a body of mass *m* moving with a velocity *v*<sub>0.</sub>
- Rod deforms under impact. Stresses reach a maximum value  $\sigma_{\max}$  and then disappear.

- To determine the maximum stress  $\sigma_{
  m max}$ 
  - Assume that the kinetic energy is transferred entirely to the structure,

$$U = \frac{1}{2} m v_0^2$$

- Assume that the stress-strain diagram obtained from a static test is also valid under impact loading.
- Energy conservation requires,

$$U = \int \frac{\sigma_{\max}^2}{2E} dV = \int \frac{P^2}{2EA} dx$$

• For the case of a uniform rod,

$$U = \frac{P^{2}L}{2EA} \Longrightarrow P = \sqrt{\frac{2UAE}{L}} = \sqrt{\frac{mv_{0}^{2}AE}{L}}$$

#### **Sample Problem**



**Horizontal Impact**: Body of mass m with velocity  $v_0$  hits the end of the nonuniform rod *BCD*. Knowing that the diameter of the portion *BC* is twice the diameter of portion *CD*, determine the maximum value of the normal stress in the rod.

Solution:

- Due to the change in diameter, the normal stress distribution is nonuniform.
- Find the static load *P* which produces the same strain energy as the impact.
- Evaluate the maximum stress resulting from the static load *P*



Solution:

Due to the change in diameter, • the normal stress distribution is nonuniform.

$$U = \frac{1}{2}mv_0^2$$
$$= \int \frac{\sigma^2}{2E} dV = \frac{P^2 L}{2EA}$$

• Find the static load *P* which produces the same strain energy as the impact.

$$U = \frac{P^{2}(L/2)}{2EA} + \frac{P^{2}(L/2)}{2E(4A)} = \frac{5}{16} \frac{P^{2}L}{EA}$$
$$P = \sqrt{\frac{16}{5} \frac{UEA}{L}} = \sqrt{\frac{8}{5} \frac{mv_{0}^{2}EA}{L}}$$

• Evaluate the maximum stress resulting from the static load P

$$\sigma_{\max} = \frac{P}{A}$$
$$= \sqrt{\frac{16}{5} \frac{UE}{AL}}$$
$$= \sqrt{\frac{8}{5} \frac{mv_0^2 E}{AL}}$$

#### **Horizontal Impact on Beams**



**Horizontal Impact**: A block of mass mmoving with a velocity  $v_0$  hits squarely the prismatic member AB at its midpoint C. Determine (a) the equivalent static load P, (b) the maximum stress  $\sigma_{max}$  in the member, and (c) the maximum deflection  $\Delta_{max}$  at C. Solution:

• Kinetic energy of the block.

$$U_m = m v_0^2 / 2$$

• Find the static load *P* which produces the same strain energy as the impact. For loaded beam as shown,

$$\begin{cases} U = \frac{1}{2} P \Delta = \frac{1}{2} \frac{P^2 L^3}{48EI} \\ P = \sqrt{\frac{96EIU}{L^3}} = \sqrt{\frac{48mv_0^2 EI}{L^3}} \end{cases}$$

• Maximum stress

$$\sigma_{\max} = \frac{M_{\max}c}{I} = \frac{PLc}{4I} = \sqrt{\frac{3mv_0^2 EI}{(I/c)^2 L}}$$

• Maximum deflection  $\Delta_{\text{max}} = \frac{PL^3}{48EI} = \sqrt{\frac{mv_0^2 L^3}{48EI}}$  0

#### **Vertical Impact on Beams**



**Vertical Impact**: A block of weight *W* is dropped from a height *h* onto the free end of the cantilever beam. Determine the maximum value of the stresses in the beam.

Solution:

• Work done by the impact load.

$$U = W(h + \Delta_d)$$

- Find the static load *P* which produces the same strain energy as the impact.
- Treat the beam as a spring:

$$U = \frac{1}{2} P \Delta_d = \frac{P^2}{2k} = \frac{1}{2} k \Delta_d^2$$

• Energy conservation requires,

$$\frac{1}{2}k\Delta_d^2 = W(h + \Delta_d)$$
$$\Rightarrow \frac{1}{2}\Delta_d^2 = \frac{W}{k}(h + \Delta_d) = \Delta_{st}(h + \Delta_d)$$
$$\Rightarrow \Delta_d^2 - 2\Delta_{st}\Delta_d - 2\Delta_{st}h = 0$$

#### **Vertical Impact on Beams**

$$\Rightarrow \Delta_d = \frac{2\Delta_{st} \pm \sqrt{\left(2\Delta_{st}\right)^2 + 8\Delta_{st}h}}{2} = \Delta_{st} \pm \sqrt{\Delta_{st}^2 + 2\Delta_{st}h} = \left(1 \pm \sqrt{1 + \frac{2h}{\Delta_{st}}}\right)\Delta_{st}$$
$$\Rightarrow \Delta_d = \left(1 + \sqrt{1 + \frac{2h}{\Delta_{st}}}\right)\Delta_{st} = K_d\Delta_{st}$$

Dynamic load factor:  $K_d = 1 + \sqrt{1 + \frac{2h}{\Delta_{st}}}$ For the present problem:

$$\Delta_{st} = \frac{WL^3}{3EI}, \quad \sigma_{st \cdot \max} = \frac{M_{\max} y_{\max}}{I} = \frac{WLc}{I}$$
$$\Rightarrow \Delta_d = K_d \Delta_{st} = \left(1 + \sqrt{1 + \frac{6EIh}{WL^3}}\right) \Delta_{st} = \frac{PL^3}{3EI}$$
$$\Rightarrow P = K_d W = \left(1 + \sqrt{1 + \frac{6EIh}{WL^3}}\right) W$$
$$\Rightarrow \sigma_{d \cdot \max} = K_{st \cdot \max} \sigma_{st} = \left(1 + \sqrt{1 + \frac{6EIh}{WL^3}}\right) \frac{WLc}{I}$$

For sudden loading 
$$(h = 0) \Rightarrow K_d = 2$$
  

$$\begin{cases}
K_d = \sqrt{\frac{2h}{\Delta_{st}}} \approx \sqrt{\frac{6EIh}{WL^3}} \\
\Delta_d = K_d \Delta_{st} \approx \sqrt{\frac{2hWL^3}{3EI}}, \\
P = K_d W \approx \sqrt{\frac{6EIhW}{L^3}} \\
\sigma_{d \cdot \max} = K_d \sigma_{st \cdot \max} \approx \sqrt{\frac{6EhWc^2}{IL}}
\end{cases}$$

#### **Vertical Impact with an Initial Velocity**



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