# **Bending Internal Forces**

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### **The Form of Internal Forces in Beams**

• Internal forces on cross-sections of bending beams







The internal forces on cross-section m-m can be simplified as *shearing forces* and *bending moments* at cross-sectional centroid.

## **Symmetric Bending**



- **Beam:** one type of prismatic bars which takes bending as the main form of deformation.
- **Symmetric longitudinal plane:** formed by the centroidal axes of cross sections.
- **Symmetric bending:** all external forces & moments acting in the symmetric longitudinal plane (beam axis being bent from a straight line to a curve within the symmetric longitudinal plane).

## **Types of Beams**



Beam with an overhang

Compound beam

## **Types of Supports and Reactions**



Fixed support Hinged or pinned support Roller support  $M_R = F_{Rx}$   $F_{Rx}$   $F_{Ry}$   $F_{Ry}$   $F_{Ry}$   $F_{Ry}$ 

#### **Internal Releases**







• Axial release

• Shear release

• Moment release



#### **Internal Releases**



## **Determinacy of Beams**

• Statically determinate beams



- Cantilever beam

• Statically indeterminate beams



- Simply supported cantilever beam



- Simply supported beam



- Continuous beam



- Overhanging beam



- Fixed beam

#### **Sign Convention of Shearing Forces and Bending Moments**

- Shearing forces
  - Positive: left up / right down (clockwise loop)



- Bending moment
  - Positive: upper half under compression / lower half under tension (top concave / bottom convex)



• Find the shearing force and bending moment at the left and right cross-section at point C.

a

A

• Solution:



2. Internal forces (assuming positive) at the right cross-section of *C*.

$$F_{SCA} = F_A = \frac{F}{2}$$
$$M_{CA} = F_A \cdot a = \frac{Fa}{2}$$



a

 $F_A = F_B$ 

R

 $F_{B}$ 

3. Internal forces (assuming positive) at the left cross-section of *C*.

 $F_{SCB} = -F_B = -\frac{F}{2}$  $M_{CB} = F_B \cdot a = \frac{Fa}{2}$ 



Note: for a concentrated force acting at *C*, the change of bending moment is zero while that of shearing force is equal to the magnitude of the concentrated force.

- Find the shearing force and bending moment at the left and right cross-section at point *C*.
- Solution:
- 1. Reaction forces at the supports

$$F_A = F_B = \frac{F}{2}$$

2. Internal force at the right cross-section of *C*.

$$F_{SCA} = F_A = \frac{F}{2}$$
$$M_{CA} = F_A \cdot a = \frac{Fa}{2}$$







3. Internal force at the left cross-section of C.  $F_{SCB} = F_B = \frac{F}{2}$   $M_{CB} = -F_B \cdot a = -\frac{Fa}{2}$   $M_{CB} = -F_B \cdot a = -\frac{Fa}{2}$   $M_{CB} = -F_B \cdot a = -\frac{Fa}{2}$ 

Note: For a concentrated moment acting at *C*, the change of shearing force is zero while that of bending moment is equal to the magnitude of the concentrated moment.

#### **Direct Calculation of Shearing Forces**

- The shearing forces at a beam cross-section is equal to the net of the external forces collected from either side of the cross-section.
- Upward net from left portion or downward net from right portion results in positive shearing force at the cross-section.
- All forces acting on a beam, including the reaction forces at the supports, must be taken into consideration.

• Directly calculate the shearing force at cross-section *C* without the explicit use of method of section.



- Solution:
  - Based on the left portion from cross-section *C*:

 $F_{\rm SC}=F_{\rm A}-F_{\rm 1}+F_{\rm 2}-F_{\rm 3}$ 

- Based on the right portion from cross-section *C*:

 $F_{\rm SC} = -F_{\rm B} + ql + F_4$ 

Note: in practice, we pick the portion with simpler loadings.

### **Direct Calculation of Bending Moments**

- The bending moment at a beam cross-section is equal to the net of the external moments with respect to the cross-section centroid, collected from either side of the cross-section
- Clockwise net from left portion or counter clockwise net from right portion results in positive bending moment.
- All forces acting on a beam, including the reaction forces at the supports, must be taken into consideration.

- Directly calculate the bending moment at cross-section *C* without the explicit use of method of section.
- Solution:
  - Based on the left portion from cross-section *C*



$$M_C = F_A d_A - F_1 d_1 - M_1$$

- Based on the right portion from cross-section C

$$M_C = -F_B d_B + F_2 d_2$$

## **Diagram of Shearing Forces & Bending Moments**

- Equation of shearing forces and bending moments
  - Equation of shearing forces:  $F_{\rm S} = F_{\rm S}(x)$
  - Equation of bending moments: M = M(x)
  - *x*: denotes the position of cross-section.
- Diagram of shearing forces and bending moments
  - Abscissa: cross-section position (*x*)
  - Ordinate: shearing forces  $(F_s)$  / bending moments (M)

• Draw the diagram of shearing forces and bending moments



• Solution:

- 1. Reaction force at the supports:  $F_A = F_B = \frac{F}{2}$
- 2. Equation of internal forces

$$F_{\rm S}(x_1) = F_A = \frac{F}{2} \qquad x_1 \in [0, a) \qquad \text{Positive: left & upward}$$
$$F_{\rm S}(x_2) = -F_B = -\frac{F}{2} \qquad x_2 \in [0, a) \qquad \text{Negative: right & upward}$$

$$M(x_{1}) = F_{A}x_{1} = \frac{Fx_{1}}{2} \quad x_{1} \in [0,a)$$
Positive: left & clockwise
$$M(x_{2}) = F_{B}x_{2} = \frac{Fx_{2}}{2} \quad x_{2} \in [0,a)$$
Positive: right & counter clockwise
$$M(x_{2}) = F_{B}x_{2} = \frac{Fx_{2}}{2} \quad x_{2} \in [0,a)$$
Positive: right & counter clockwise

• Draw the diagram of shearing forces and bending moments



- Solution:
- 1. Reaction forces at the supports

$$F_A = F_B = \frac{F}{2}$$

2. Equations of internal forces  $F_{\rm S}(x_1) = \frac{F}{2}; \quad F_{\rm S}(x_2) = \frac{F}{2}$   $M(x_1) = \frac{Fx_1}{2}; \quad M(x_2) = -\frac{Fx_2}{2}$ 



0.5F

F<sub>S</sub>,



- Plot the diagram of shearing forces and bending moments
- Solution:
  - 1. Reaction forces at the supports





$$F_{s}(x) + q(x)dx - F_{s}(x) - dF_{s}(x) = 0$$
  

$$\Rightarrow q(x) = \frac{dF_{s}(x)}{dx}, \qquad F_{s2} - F_{s1} = \int_{x_{1}}^{x_{2}} q(x) dx$$
  

$$M(x) + dM(x) - \frac{1}{2}q(x)d^{2}x - M(x) - F_{s}(x)dx = 0$$
  

$$\Rightarrow F_{s}(x) = \frac{dM(x)}{dx}, \qquad M_{2} - M_{1} = \int_{x_{1}}^{x_{2}} F_{s}(x)dx$$
  

$$\Rightarrow q(x) = \frac{dM^{2}(x)}{dx^{2}}$$

Sign convention of q(x): up positive / down negative.

• Without distributed load: q(x) = 0

$$q(x) = \frac{dF_s(x)}{dx} = 0 \implies F_s(x) = \text{const}$$

Note: the slope of the diagram of shearing forces is zero.

$$\frac{dM(x)}{dx} = F_s(x) = const \implies M(x) = (const) \times x + (const)$$

Note: the slope of the diagram of bending moments is constant.

• With uniformly distributed load: q(x) = const

$$q(x) = \frac{dF_s(x)}{dx} = \text{const} \implies F_s(x) = (\text{const}) \times x + (\text{const})$$

Note: the slope of the diagram of shearing forces is constant.

$$\frac{dM(x)}{dx} = F_S(x) \implies$$
$$M(x) = (\text{const}) \times x^2 + (\text{const}) \times x + (\text{const})$$

Note: M(x) is a second order polynomial of x.



• Draw the diagram of shearing forces and bending moments.



- Draw the diagram of shearing forces and bending moments.
- Solution:

$$F_A = \frac{3ql}{8}$$
$$F_B = \frac{ql}{8}$$



- Draw the diagram of shearing forces and bending moments.
- Solution:

 $F_A = qa$  $F_B = qa$ 



- Draw the diagram of shearing forces and bending moments.
- Solution:

$$F_A = \frac{qa}{2}$$
$$F_B = \frac{qa}{2}$$



## **Diagram of Internal Forces for Plane Frames**

- Plane frames are composed of prismatic bars with different orientations in the same plane.
- Neighboring bars are connected via rigid joints
- Internal forces acting on cross-sections of component bars include axial force, shearing force, and bending moment.



## **Diagram of Internal Forces for Plane Frames**

- Sign convention
  - Axial forces: tension positive / compression negative
  - Shearing forces and bending moments: the same sign convention when the plane frame is observed from inside.
- Drawing convention
  - Axial forces and shearing forces: draw at either side of the bar with sign labeled.
  - Bending moments: draw at the side under tension without sign labeled.

- Draw the diagram of axial forces, shearing forces and bending moments for the plane frame shown below.
- Solution:
- 1. reaction forces at the supports.

$$\sum M_{A} = 0 \Longrightarrow$$

$$F_{B} = \frac{\left(8 \times 1 + 1 \times 4^{2} / 2 - 1 \times 1\right)}{3} = 5 \text{ kN}$$

$$\sum_{1 \times 4} F_x = 0 \Longrightarrow$$
$$1 \times 4 + F_{Ax} - 1 = 0 \Longrightarrow F_{Ax} = -3 \text{ kN}$$

$$\sum F_{y} = 0 \Longrightarrow F_{Ay} + F_{B} - 8 = 0$$
$$\implies F_{Ay} = 3 \text{ kN}$$



- 2. Diagram of internal forces1) Diagram of axial forces
  - $BC: F_{N1} = -5 \text{ kN}$  $DC: F_{N2} = -1 \text{ kN}$  $AD: F_{N3} = -3 \text{ kN}$





- 2) Diagram of shearing forces
  - BC: horizontal line
  - DC: horizontal line
  - DA: oblique straight line









## **Diagram of Internal Forces for Curved Beams**

- Draw the diagram of axial forces, shearing forces and bending moments for the curved beam shown below.
- Solution: • Static equilibrium  $F_{\rm N}(\theta) = F \sin \theta$   $F_{\rm S}(\theta) = -F \cos \theta$   $M(\theta) = Fy = FR \sin \theta$ FRΟ Ð Ð Θ  $F_{\rm S}$ B MFF46

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